

# A variational approach to moving contact line hydrodynamics

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(Received 12 February 2006 and in revised form 23 April 2006)

In immiscible two-phase flows, the contact line denotes the intersection of the fluid–fluid interface with the solid wall. When one fluid displaces the other, the contact line moves along the wall. A classical problem in continuum hydrodynamics is the incompatibility between the moving contact line and the no-slip boundary condition, as the latter leads to a non-integrable singularity. The recently discovered generalized Navier boundary condition (GNBC) offers an alternative to the no-slip boundary condition which can resolve the moving contact line conundrum. We present a variational derivation of the GNBC through the principle of minimum energy dissipation (entropy production), as formulated by Onsager for small perturbations away from equilibrium. Through numerical implementation of a continuum hydrodynamic model, it is demonstrated that the GNBC can quantitatively reproduce the moving contact line slip velocity profiles obtained from molecular dynamics simulations. In particular, the transition from complete slip at the moving contact line to near-zero slip far away is shown to be governed by a power-law partial-slip regime, extending to mesoscopic length scales. The sharp (fluid–fluid) interface limit of the hydrodynamic model, together with some general implications of slip versus no slip, are discussed.

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## 1. Introduction

The no-slip boundary condition states that there can be no relative motion at the fluid–solid interface (Batchelor 1967). It is generally regarded as a cornerstone in continuum hydrodynamics, owing to its proven applicability in diverse fluid-flow problems. However, decades ago it was discovered that in immiscible two-phase flows, the moving contact line (MCL), defined as the intersection of the fluid–fluid interface with the solid wall, is incompatible with the no-slip boundary condition (Moffatt 1964; Huh & Scriven 1971; Dussan V. & Davis 1974; Dussan V. 1976, 1979; de Gennes 1985). As shown by Dussan V. & Davis (1974), under usual hydrodynamic assumptions, namely the incompressible Newtonian fluids, no-slip boundary condition, and smooth rigid solid walls, there is a velocity discontinuity at the MCL, and the tangential force exerted by the fluids on the solid bounding surface in the vicinity of the MCL is infinite. This is the well-known contact-line singularity. In the past two decades, it has been shown through molecular dynamics (MD) simulations that near-complete slip indeed occurs at the MCL (Koplik, Banavar & Willemsen 1988,

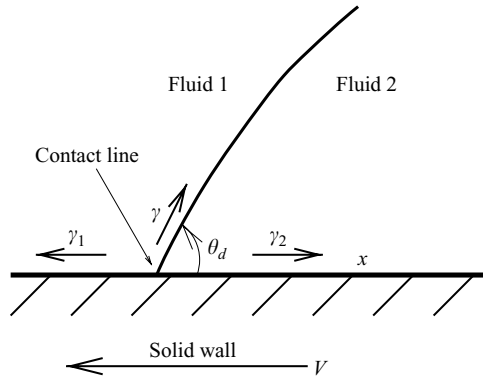


FIGURE 1. When one fluid displaces another immiscible fluid, the contact line is moving relative to the solid wall. Owing to the contact-line motion, the dynamic contact angle  $\theta_d$  deviates from the static contact angle  $\theta_s$ , which is determined by the Young equation  $\gamma \cos \theta_s + \gamma_2 = \gamma_1$ .

1989; Thompson & Robbins 1989; Thompson, Brinckerhoff & Robbins 1993). This finding presented a conundrum for classical hydrodynamics, due to a lack of viable alternatives apart from *ad hoc* fixes. Furthermore, in the absence of a viable boundary condition which can reproduce the MD results, an accurate continuum description of immiscible flows at the micro- or nanoscales remained an elusive goal.

Through analysis of extensive MD data, it was recently discovered that there is indeed a differential boundary condition, denoted the generalized Navier boundary condition (GNBC), which resolves the MCL conundrum (Qian, Wang & Sheng 2003). Here we show that the GNBC can be derived variationally from the principle of minimum energy dissipation (Onsager 1931*a, b*), and its implementation through the use of a Cahn–Hilliard (CH) free energy functional (Cahn & Hilliard 1958) leads to quantitative predictions in excellent agreement with MD simulation results. In what follows, the MCL problem is briefly recapitulated in §2. The variational derivation of the GNBC in §3 is followed by a numerical demonstration of its consequences in §4. It is shown that the transition from near-complete slip at the MCL to near-zero slip (no-slip) far away from the MCL is not confined to a molecular-sized region around the MCL. Instead, the transition follows a power-law profile of partial slipping, extending to mesoscopic scales (Qian, Wang & Sheng 2004). The sharp/diffuse (fluid–fluid) interface limits of our theory and associated issues are described in §5. In §6 we discuss some general implications of replacing the no-slip boundary condition, which can be regarded as an approximation to the GNBC in single-phase flow regions, by the more accurate GNBC. In particular, it is argued that slip and partial-slip boundary conditions offer the prospect of nanoscale interface engineering to ‘tune’ the amount of slipping.

## 2. Recapitulation of the moving contact line problem

Consider an immiscible two-phase flow where one fluid displaces the other (see figure 1). If the no-slip boundary condition is applied along the wall, it can be shown that the tangential viscous stress varies as  $\eta V/x$ , where  $\eta$  is the viscosity,  $V$  is the wall speed in the reference frame where the fluid–fluid interface is stationary, and  $x$  is the distance along the wall away from the MCL. This variation leads to diverging stress as  $x \rightarrow 0$ . In particular, this stress divergence is non-integrable and implies infinite (viscous) dissipation (Dussan V. & Davis 1974). Over the years

there have been numerous models and proposals aiming to resolve this problem for example, the kinetic adsorption/desorption model by Blake & Haynes (1969), the slip models by Hocking (1977), Huh & Mason (1977), and Zhou & Sheng (1990), the interface formation/disappearance model by Shikhmurzaev (1997), the diffuse-interface models by Seppecher (1996), Jacqmin (2000), Chen, Jasnow & Vinals (2000), Pismen & Pomeau (2000), and Briant & Yeomans (2004), and the variational model by Glasner (2005).

In another approach to the MCL problem, Cox (1986) carried out an asymptotic analysis and found, to the leading order in the capillary number  $Ca = \eta V/\gamma$ , a relation for the dependence of the apparent contact angle (the angle of the fluid–fluid interface at some mesoscopic distance away from the fluid–solid interface) on the microscopic contact angle, the capillary number, and the distance from the contact line over which slip occurs. In this approach, the details of the slip flow in the inner region around the MCL are absorbed into a model-dependent constant in the asymptotic relation. In particular, it has been shown that the asymptotic behaviour is independent of the microscopic boundary condition(s) (Dussan V. 1976). But this conclusion only deepens the mystery of what happens at the contact line.

In the past two decades, MD simulations have shown that near-complete slip indeed occurs at the MCL (Koplik *et al.* 1988, 1989; Thompson & Robbins 1989; Thompson *et al.* 1993). A small amount of partial slipping was also observed in single-phase flows (Thompson & Robbins 1990; Thompson & Troian 1997; Barrat & Bocquet 1999a; Cieplak, Koplik & Banavar 2001). Such partial slip can be accounted for by the Navier boundary condition (NBC), proposed nearly two centuries ago by Navier (1823):  $v^{slip} = l_s \dot{\gamma}$ , where  $v^{slip}$  is the slip velocity at the surface, measured relative to the (moving) wall,  $l_s$  is the slip length, and  $\dot{\gamma}$  is the shear rate at the surface. The small value of  $l_s$  explains why the NBC is practically indistinguishable from the no-slip boundary condition in single-phase macroscopic flows.

While the NBC can account for the small amount of slip in high-shear-rate single-phase flows, it fails to account, by an order of magnitude, for the near-complete slip at the MCL (Thompson & Robbins 1989; Thompson *et al.* 1993). Recently, the MD–continuum hybrid simulation has been applied as a tool to investigate this problem by Hadjiconstantinou (1999), and by Ren & E (2005a). But such approaches leave unresolved the problem of the MCL boundary condition. Lack of a viable boundary condition implies that an accurate MCL description (and hence immiscible two-phase flows) can only be attained through MD simulations, in systems far too small compared with most of the experimentally achievable samples.

The recent formulation of the GNBC (Qian *et al.* 2003) resolves the MCL conundrum. The GNBC states that the slip velocity is proportional to the total tangential stress – the sum of the viscous stress and the uncompensated Young stress; the latter arises from the deviation of the fluid–fluid interface from its static configuration. Here we show the GNBC to be derivable from the principle of minimum energy dissipation. Its form is hence uniquely determined by the thermodynamics of immiscible two-phase flows.

### 3. Variational derivation of moving contact line hydrodynamics

#### 3.1. The principle of minimum energy dissipation

There is a minimum dissipation theorem for incompressible single-phase flows. According to Batchelor (1967) (attributable to Helmholtz), ‘the rate of dissipation in the flow in a given region with negligible inertia forces is less than that in any

other solenoidal velocity distribution (of zero divergence) in the same region with the same values of the velocity at all points of the boundary of the region.' The variational principles involving dissipation have been further developed in the works of Rayleigh (1873), Onsager (1931*a, b*), Edwards & Freed (1974), and Doi (1983). In mathematical terms, if we ignore inertia forces for the moment (they can be added in at the end, see equation (3.36)) and let the variables  $\alpha_1, \dots, \alpha_n$  describe the displacement from thermodynamic equilibrium,  $\dot{\alpha}_1, \dots, \dot{\alpha}_n$  being the corresponding rates and  $F(\alpha_1, \dots, \alpha_n)$  the free energy, then for a set of simultaneous irreversible processes governed by

$$\sum_{j=1}^n \rho_{ij} \dot{\alpha}_j = - \frac{\partial F(\alpha_1, \dots, \alpha_n)}{\partial \alpha_i} \quad (i = 1, \dots, n), \quad (3.1)$$

where the coefficients  $\rho_{ij}$  are introduced through the linear relations between the rates  $\{\dot{\alpha}_i\}$  and the 'forces'  $\{-\partial F/\partial \alpha_i\}$ , a variational principle (Onsager 1931*a, b*) can be formulated, namely

$$\delta[\Phi(\dot{\alpha}, \dot{\alpha}) + \dot{F}(\alpha, \dot{\alpha})] = \sum_{i=1}^n \left( \frac{\partial \Phi}{\partial \dot{\alpha}_i} + \frac{\partial F}{\partial \alpha_i} \right) \delta \dot{\alpha}_i = 0, \quad (3.2)$$

where  $\Phi(\dot{\alpha}, \dot{\alpha})$  is the dissipation function defined by

$$\Phi(\dot{\alpha}, \dot{\alpha}) \equiv \frac{1}{2} \sum_{i,j} \rho_{ij} \dot{\alpha}_i \dot{\alpha}_j, \quad (3.3)$$

and  $\dot{F}(\alpha, \dot{\alpha})$  is the rate of change of the free energy:

$$\dot{F}(\alpha, \dot{\alpha}) \equiv \sum_{i=1}^n \frac{\partial F(\alpha_1, \dots, \alpha_n)}{\partial \alpha_i} \dot{\alpha}_i. \quad (3.4)$$

If  $\{\alpha_i\}$  and  $\{\dot{\alpha}_i\}$  are field variables (defined at every spatial point), the summation in equation (3.2) naturally becomes integral and the partial derivatives with respect to  $\alpha_i$  and  $\dot{\alpha}_i$  are replaced by functional derivatives. Microscopic reversibility requires the reciprocal relations  $\rho_{ij} = \rho_{ji}$ . We note that the variation in equation (3.2) should be taken *with respect to the rates*  $\{\dot{\alpha}_i\}$  for prescribed  $\{\alpha_i\}$ , and the extremum given by equation (3.2) is always a minimum because  $\Phi(\dot{\alpha}, \dot{\alpha})$  is quadratic in  $\{\dot{\alpha}_i\}$  and positive definite (Onsager 1931*b*). The dissipation function  $\Phi$  as given by equation (3.3) is half the rate of energy dissipation, owing to the required consistency between the 'force balance' condition (3.1), the minimum condition (3.2), and the fact that the dissipative forces are linear in rates. It can be shown that the principle of minimum energy dissipation yields the most probable course of an irreversible process, provided the displacements from thermodynamic equilibrium are small (Onsager 1931*b*; Onsager & Machlup 1953). We note that the variational principle presented here is a special form of the principle of minimum energy dissipation formulated by Onsager. The reason that we use the free energy (instead of the entropy as Onsager originally did) to formulate the variational principle is that in the present problem, conditions of mechanical equilibrium necessarily enter the laws of irreversible processes (Onsager 1931*b*; Doi 1983). The consequences are: (i) the dissipation function  $\Phi$  defined here differs from that by Onsager by a factor  $T$ , the temperature which is assumed to be uniform in the fluids; (ii) the rate of change of the free energy  $\dot{F}$  equals  $-T\dot{S} + \dot{W}$ , where  $S$  and  $W$  denote the entropy and work, respectively. This point will be

elaborated after equation (3.24). In Appendix A we use a single-variable case to illustrate the underlying physics of the principle of minimum energy dissipation.

### 3.2. Single-phase flow

When applied to a single-phase flow confined by solid surfaces, the variational principle in equation (3.2) becomes

$$\delta\Phi(\dot{\alpha}, \dot{\alpha}) = \sum_{i=1}^n \frac{\partial\Phi}{\partial\dot{\alpha}_i} \delta\dot{\alpha}_i = 0. \quad (3.5)$$

Here the rates  $\{\dot{\alpha}_i\}$  correspond to the velocity field  $\mathbf{v}(\mathbf{r})$  and the  $\dot{F}(\alpha, \dot{\alpha})$  term defined by equation (3.4) drops out in the variation because the free energy  $F(\{\alpha_i\})$  is constant in single-phase flows. As the dissipation function  $\Phi(\dot{\alpha}, \dot{\alpha})$  equals half the rate of energy dissipation, equation (3.5) leads directly to the minimum dissipation theorem by Helmholtz (Batchelor 1967). That is, once the values of the velocity are prescribed at the solid surfaces, the rate of viscous dissipation is minimized by the solution of the Stokes equation. Physically, when fluid slipping occurs at the solid surface, there is dissipation at the fluid–solid interface as well. Below we show that to minimize the total rate of energy dissipation  $R_1[\mathbf{v}]$  ( $= 2\Phi(\dot{\alpha}, \dot{\alpha})$ ), the incompressible flow must satisfy the Stokes equation and the NBC simultaneously. Mathematically, this means that the Stokes equation

$$-\nabla p + \eta\nabla^2\mathbf{v} = 0, \quad (3.6)$$

and the NBC

$$\beta v_\tau^{slip} = -\sigma_{n\tau}^{visc} = -\eta(\partial_n v_\tau + \partial_\tau v_n), \quad (3.7)$$

can be derived by minimizing the functional

$$R_1[\mathbf{v}] = \int d\mathbf{r} \left[ \frac{\eta}{2} (\partial_i v_j + \partial_j v_i)^2 \right] + \int dS [\beta (v_\tau^{slip})^2], \quad (3.8)$$

with respect to the velocity distribution. Here  $p$  is the pressure which plays the role of the Lagrange multiplier for the incompressibility condition  $\nabla \cdot \mathbf{v} = 0$ ,  $\eta$  is the shear viscosity,  $\beta$  is the slip coefficient, subscript  $n$  denotes the outward surface normal, subscript  $\tau$  denotes the direction tangential to the surface,  $v_\tau^{slip}$  is the slip velocity, defined as the tangential fluid velocity at the solid surface, measured relative to the (moving) wall,  $\sigma_{n\tau}^{visc}$  is the  $n\tau$  component of the Newtonian viscous stress tensor, and  $\int dS$  denotes integration over the solid surface.

The functional  $R_1[\mathbf{v}]$  equals  $2\Phi(\dot{\alpha}, \dot{\alpha})$  for single-phase flows, measuring the total rate of dissipation due to viscosity in the bulk and slipping at the solid surface. As the fluids and solid are all composed of interacting molecules with different interaction potentials, it is reasonable to assume that the qualitative form of the energy dissipation at the fluid–solid interface is similar to the viscous dissipation in the bulk. That is, if physically the rate of viscous dissipation in the bulk, hereafter denoted by  $R_v$ , arises from the internal friction when there is a non-zero rate of strain, then there should be a rate of dissipation at the fluid–solid interface, hereafter denoted by  $R_s$ , that arises from surface friction between the fluid and solid when there is non-zero slip. Hence if we start with the expression  $R_v = \int d\mathbf{r} [\eta(\partial_i v_j + \partial_j v_i)^2/2]$  for the bulk viscous dissipation and assume a simple shear flow with shear rate  $\partial_z v_x \neq 0$  only, then  $R_v = \int d\mathbf{r} [\eta(\partial_z v_x)^2]$ . For a fluid boundary layer of thickness  $h$  (on the order of fluid–solid molecular interaction range) along the  $z$ -direction (normal to the solid surface),  $R_v$  becomes  $R_v = \int dS [(\eta/h)(\Delta v_x)^2]$ , where  $\Delta v_x = h\partial_z v_x$  is the variation of

the tangential velocity  $v_x$  across the boundary layer and the volume integral  $\int d\mathbf{r}[\cdot]$  is transformed into the surface integral  $h \int dS[\cdot]$ . Based on similarity considerations, the form of  $R_v = \int dS[(\eta/h)(\Delta v_x)^2]$  directly suggests  $R_s = \int dS[\beta(v_\tau^{slip})^2]$  because as the counterpart of  $\Delta v_x$ , the slip velocity  $v_\tau^{slip}$  is the tangential velocity difference across the fluid–solid interface, and the slip coefficient  $\beta$  has the same dimension as  $\eta/h$ . The fact that the fluid–solid interaction is different from the fluid–fluid molecular interaction means that  $\beta$ , which measures the viscous coupling at the fluid–solid interface, may be used to define a slip length  $l_s$  as  $l_s = \eta/\beta$ . This form of  $R_s$  is positive definite and quadratic in  $v_\tau^{slip}$ , hence in accordance with the general form of a dissipation function. We write the single-phase flow dissipation  $R_1$  as the sum of  $R_v$  due to viscosity and  $R_s$  due to slipping:  $R_1 = R_v + R_s$ , with

$$R_v[\mathbf{v}] = \int d\mathbf{r} \left[ \frac{\eta}{2} (\partial_i v_j + \partial_j v_i)^2 \right] \quad (3.9)$$

and

$$R_s[\mathbf{v}] = \int dS [\beta (v_\tau^{slip})^2]. \quad (3.10)$$

Associated with the variation of the velocity field  $\mathbf{v}(\mathbf{r}) \rightarrow \mathbf{v}(\mathbf{r}) + \delta\mathbf{v}(\mathbf{r})$ , the change in  $R_v$  is given by

$$\delta R_v = -2\eta \int d\mathbf{r} [\partial_j (\partial_j v_i + \partial_i v_j) \delta v_i] + 2\eta \int dS [\partial_n v_\tau \delta v_\tau + \partial_\tau v_n \delta v_\tau], \quad (3.11)$$

and that in  $R_s$  given by

$$\delta R_s = 2\beta \int dS [v_\tau^{slip} \delta v_\tau]. \quad (3.12)$$

Imposing the incompressibility condition  $\partial_i v_i = 0$  by the use of a Lagrange multiplier  $\alpha(\mathbf{r})$  leads to one more term  $\int d\mathbf{r} [\alpha \partial_i v_i]$ , with variation given by

$$\delta \int d\mathbf{r} [\alpha \partial_i v_i] = - \int d\mathbf{r} [\partial_i \alpha \delta v_i]. \quad (3.13)$$

Here the boundary condition  $v_n = 0$  has been used at the solid surface, where only  $\delta v_\tau$  is allowed. From equations (3.11), (3.12), and (3.13), we obtain the Euler–Lagrange equations

$$-2\eta \partial_j (\partial_j v_i + \partial_i v_j) - \partial_i \alpha = 0 \quad (3.14)$$

in the bulk and

$$2\eta (\partial_n v_\tau + \partial_\tau v_n) + 2\beta v_\tau^{slip} = 0 \quad (3.15)$$

at the surface. Note that equation (3.14) is identical to the Stokes equation (3.6) with  $\partial_j v_j = 0$  and  $\alpha = -2p$ , and equation (3.15) reduces to the NBC (3.7). That equations (3.6) and (3.7) can both be derived variationally from the minimization of  $R_1$  is a generalization of the minimum dissipation theorem by taking into account fluid slipping at the solid surface.

It should be emphasized that while  $R_s$  arises from the assumption of fluid–solid interface slipping, there is no specification of how much slipping there should be. In other words, even an infinitesimal amount of interface slipping would lead to equations (3.6) and (3.7). In particular, the no-slip boundary condition is obtained in the limit of  $\beta \rightarrow \infty$ , corresponding to a vanishing slip length  $l_s = \eta/\beta$ . In the other limit,  $\beta \rightarrow 0$ , we would have only the first term on the right-hand side of

equation (3.8), and  $\sigma_{n\tau}^{visc} = 0$  on the boundary. Thus the NBC interpolates between the zero-tangential-viscous-stress limit and the no-slip limit.

### 3.3. Immiscible two-phase flow

To generalize the functional  $\Phi(\dot{\alpha}, \dot{\alpha}) = \frac{1}{2} R_1 [\mathbf{v}]$  from single-phase to immiscible two-phase flows, it is recognized that a free energy functional is required to stabilize the interface separating the two immiscible fluids. Hence the introduction of a Landau free energy functional  $\mathcal{F}[\phi(\mathbf{r})]$  is a necessity, assumed to have the form (Cahn & Hilliard 1958; Bray 1994)

$$\mathcal{F}[\phi(\mathbf{r})] = \int d\mathbf{r} \left[ \frac{K}{2} (\nabla\phi)^2 + f(\phi) \right], \quad (3.16)$$

where  $K$  is a positive material parameter and the potential  $f(\phi)$  has a double-well structure. This structure is required to stabilize the fluid–fluid interface between the two stable fluid phases  $\phi_{\pm}$  (where  $\partial f(\phi_{\pm})/\partial\phi = 0$  and  $\partial^2 f(\phi_{\pm})/\partial\phi^2 > 0$ ). Given the particular form of  $f(\phi)$ , the finite interfacial thickness and interfacial tension can be obtained by minimizing  $\mathcal{F}[\phi(\mathbf{r})]$  subject to suitable boundary conditions for an interface. Here the phase field  $\phi(\mathbf{r})$  measures the (conserved) composition locally defined by  $\phi = (\rho_2 - \rho_1)/(\rho_2 + \rho_1)$ , with  $\rho_1$  and  $\rho_2$  being the number densities of the two fluid species. Similar definitions of the order parameter  $\phi$  in terms of densities may be found in Jasnow & Vinals (1996), Kuksenok, Jasnow & Balazs (2003), and Briant & Yeomans (2004) for immiscible two-phase flows. We also introduce the interfacial free energy per unit area at the fluid–solid interface,  $\gamma_{fs}(\phi)$ , which is a function of the local composition. Two quantities  $\mu$  and  $L$  can be defined from the variation of the total free energy

$$F = \mathcal{F}[\phi] + \int dS[\gamma_{fs}(\phi)],$$

that is,

$$\delta \left\{ \mathcal{F}[\phi] + \int dS[\gamma_{fs}(\phi)] \right\} = \int d\mathbf{r} [\mu\delta\phi] + \int dS[L\delta\phi], \quad (3.17)$$

in which  $\mu = \delta\mathcal{F}/\delta\phi = -K\nabla^2\phi + \partial f(\phi)/\partial\phi$  by definition is the chemical potential in the bulk and  $L$ , given by  $K\partial_n\phi + \partial\gamma_{fs}(\phi)/\partial\phi$ , is the corresponding quantity at the solid surface. Minimizing the total free energy with respect to  $\phi$  yields the equilibrium conditions  $\mu = C$  in the bulk and  $L = 0$  at the surface,  $C$  being a constant acting as the Lagrange multiplier for the conservation of  $\phi$ . It will be shown below that  $L = 0$  leads to the Young equation for the static contact angle.

From the equilibrium conditions derived above, we see that deviations from the two-phase equilibrium may be measured by the ‘forces’  $\nabla\mu$  in the bulk and  $L$  at the fluid–solid interface. For small perturbations away from the equilibrium, the additional rate of dissipation  $R_{\phi}$  arises from system responses that are linear in  $\nabla\mu$  and  $L$ . Such responses are described by the diffusive current  $\mathbf{J}$  in the bulk and the material time derivative of  $\phi$  at the solid surface, i.e.  $\dot{\phi} = \partial\phi/\partial t + v_{\tau}\partial_{\tau}\phi$ . The conservation of  $\phi$  means that the diffusive current and the material time derivative of  $\phi$  satisfy the continuity equation

$$\dot{\phi} \equiv \frac{\partial\phi}{\partial t} + \mathbf{v} \cdot \nabla\phi = -\nabla \cdot \mathbf{J}. \quad (3.18)$$

But at the fluid–solid interface, diffusive transport normal to the interface is possible ( $\partial_n J_n \neq 0$  in general), hence the interfacial  $\phi$  is not conserved. In anticipation of

the relevant dynamics governing the conserved  $\phi$  in the bulk and non-conserved interfacial  $\phi$ ,  $\mathbf{J}$  in the bulk and  $\dot{\phi}$  at the solid surface are hence the two additional rates associated with the coexistence of two phases. The additional rate of dissipation  $R_\phi$  due to the displacement from the two-phase equilibrium may be constructed as a functional quadratic in the rates,  $R_\phi = R_d + R_r$ , where

$$R_d = \int d\mathbf{r} \left[ \frac{\mathbf{J}^2}{M} \right], \quad (3.19)$$

and

$$R_r = \int dS \left[ \frac{\dot{\phi}^2}{\Gamma} \right], \quad (3.20)$$

with  $M$  and  $\Gamma$  introduced as two phenomenological parameters. Combining  $R_\phi$  with  $R_1$  in equation (3.8), we obtain the dissipation function  $\Phi(\dot{\alpha}, \dot{\alpha})$  for immiscible two-phase flows:

$$\Phi = \int d\mathbf{r} \left[ \frac{\eta}{4} (\partial_i v_j + \partial_j v_i)^2 \right] + \int dS \left[ \frac{\beta}{2} (v_\tau^{slip})^2 \right] + \int d\mathbf{r} \left[ \frac{\mathbf{J}^2}{2M} \right] + \int dS \left[ \frac{\dot{\phi}^2}{2\Gamma} \right], \quad (3.21)$$

which equals half the total rate of energy dissipation  $R_2$  in two-phase flows, i.e.

$$\Phi = \frac{1}{2} R_2 = \frac{1}{2} (R_1 + R_\phi) = \frac{1}{2} (R_v + R_s + R_d + R_r). \quad (3.22)$$

Now the viscosity  $\eta$  and slip coefficient  $\beta$  in equation (3.21), respectively, are understood to take on their respective values for the two immiscible fluids on the two sides of the interface. Note that the right-hand side of equation (3.21) consists of four terms, contributed by the four physically distinct sources of dissipation – the shear viscosity in the bulk, the fluid slipping at the solid surface, the composition diffusion in the bulk, and the composition relaxation at the solid surface. In addition, each term that contributes to  $\Phi$  is positive definite and quadratic in a rate that arises from the displacement from the equilibrium and accounts for a particular source of dissipation. This quadratic dependence follows the general rule governing entropy production in a thermodynamic process (Landau & Lifshitz 1997); it directly arises from the linear response to a small perturbation away from the equilibrium.

The rate of change of the free energy, i.e.  $\dot{F}(\dot{\alpha}, \dot{\alpha})$  in equation (3.4), may be written as

$$\dot{F} = \int d\mathbf{r} \left[ \mu \frac{\partial \phi}{\partial t} \right] + \int dS \left[ L \frac{\partial \phi}{\partial t} \right], \quad (3.23)$$

in accordance with the variation of the total free energy in equation (3.17). Substituting  $\partial \phi / \partial t = \dot{\phi} - \mathbf{v} \cdot \nabla \phi$  into equation (3.23) and using  $\int d\mathbf{r} [\mu \dot{\phi}] = \int d\mathbf{r} [-\mu \nabla \cdot \mathbf{J}] = \int d\mathbf{r} [\nabla \mu \cdot \mathbf{J}]$  with  $\int d\mathbf{r} [\nabla \cdot (\mu \mathbf{J})] = \int dS [\mu J_n] = 0$  because of the impermeability condition  $J_n = 0$  at the solid surface, we obtain

$$\dot{F} = \int d\mathbf{r} [\nabla \mu \cdot \mathbf{J} - \mu \mathbf{v} \cdot \nabla \phi] + \int dS [L(\dot{\phi} - v_\tau \partial_\tau \phi)]. \quad (3.24)$$

Note that the laws of thermodynamics require  $\dot{F} = -T\dot{S} + \dot{W}$ , where  $S$  and  $W$  denote the entropy and work, respectively. Here  $F$  is the free energy associated with the composition field  $\phi$ , consequently the entropy part  $-T\dot{S}$  must arise from the composition diffusion (in the bulk) and relaxation (at the fluid–solid interface) while the work rate  $\dot{W}$  is due to the work done by the flow to the fluid–fluid interface. That



is

$$-T\dot{S} = \int d\mathbf{r}[\nabla\mu \cdot \mathbf{J}] + \int dS[L\dot{\phi}], \quad (3.25)$$

and

$$\dot{W} = \int d\mathbf{r}[-\mathbf{v} \cdot (\mu\nabla\phi)] + \int dS[-v_\tau(L\partial_\tau\phi)]. \quad (3.26)$$

It will be seen that  $\mu\nabla\phi$  and  $L\partial_\tau\phi$  are the ‘elastic’ force/stress exerted by the interface on the flow. It is clear that in the steady state  $\dot{F}=0$  because the work is fully transformed into entropy, i.e.  $\dot{W}=T\dot{S}$ . This should be obvious since in the steady state the diffusive transport of the fluid–fluid interface is balanced by the kinematic transport by the flow, hence its free energy is invariant over time.

Therefore, for immiscible two-phase flows the variational principle in equation (3.2) may be expressed by using the functional

$$\begin{aligned} \Phi + \dot{F} = & \int d\mathbf{r} \left[ \frac{\eta}{4} (\partial_i v_j + \partial_j v_i)^2 \right] + \int dS \left[ \frac{\beta}{2} (v_\tau^{slip})^2 \right] + \int d\mathbf{r} \left[ \frac{\mathbf{J}^2}{2M} \right] + \int dS \left[ \frac{\dot{\phi}^2}{2\Gamma} \right] \\ & + \int d\mathbf{r} [\nabla\mu \cdot \mathbf{J} - \mu\mathbf{v} \cdot \nabla\phi] + \int dS [L(\dot{\phi} - v_\tau\partial_\tau\phi)]. \end{aligned} \quad (3.27)$$

Based on equation (3.27), a hydrodynamic model for the contact-line motion can be derived by minimizing  $\Phi + \dot{F}$  with respect to the rates  $\{\mathbf{v}, \mathbf{J}, \dot{\phi}\}$ , supplemented with the incompressibility condition  $\nabla \cdot \mathbf{v} = 0$ .

As  $\Phi$  is quadratic in  $(\mathbf{J}, \dot{\phi})$  and  $\dot{F}$  is linear in  $(\mathbf{J}, \dot{\phi})$ , the Euler–Lagrange equation with respect to  $\mathbf{J}$  is given by

$$\mathbf{J} = -M\nabla\mu, \quad (3.28)$$

where the parameter  $M$  introduced in equation (3.19) is seen to have the meaning of a mobility coefficient. Substituting equation (3.28) into the continuity equation (3.18) for  $\phi$  yields the anticipated advection–diffusion equation

$$\dot{\phi} = \frac{\partial\phi}{\partial t} + \mathbf{v} \cdot \nabla\phi = -\nabla \cdot \mathbf{J} = M\nabla^2\mu. \quad (3.29)$$

Similarly, the corresponding Euler–Lagrange equation for minimizing  $\Phi + \dot{F}$  with respect to  $\dot{\phi}$  at the solid surface is

$$\dot{\phi} = \frac{\partial\phi}{\partial t} + v_\tau\partial_\tau\phi = -\Gamma L(\phi). \quad (3.30)$$

That is, at the fluid–solid interface, the relaxation dynamics of the interfacial  $\phi$  is linear in  $L(\phi)$ , i.e. Allen–Cahn dynamics for non-conserved quantities (Bray 1994).

Now we show that the Stokes equation with the capillary force,

$$-\nabla p + \eta\nabla^2\mathbf{v} + \mu\nabla\phi = 0, \quad (3.31)$$

and the GNBC with the uncompensated Young stress,

$$\beta(\phi)v_\tau^{slip} = -\eta(\partial_n v_\tau + \partial_\tau v_n) + L(\phi)\partial_\tau\phi, \quad (3.32)$$

can be obtained by minimizing  $\Phi + \dot{F}$  with respect to the fluid velocity. Here  $\mu\nabla\phi$  is the capillary force density (Chella & Vinals 1996; Qian *et al.* 2003),  $\beta(\phi)$  is the slip coefficient which may locally depend on the composition, and  $L(\phi)\partial_\tau\phi$  is the uncompensated Young stress which vanishes at equilibrium (Qian *et al.* 2003). From equations (3.21), (3.22), (3.24) and (3.27), we see that the dependence of  $\Phi + \dot{F}$  on the velocity comes from  $\frac{1}{2}R_1 = \frac{1}{2}(R_v + R_s)$  in  $\Phi$  and  $\int d\mathbf{r}[-\mu\mathbf{v} \cdot \nabla\phi] + \int dS[-Lv_\tau\partial_\tau\phi]$

in  $\dot{F}$ . Consider a variation of the velocity field  $\mathbf{v}(\mathbf{r}) \rightarrow \mathbf{v}(\mathbf{r}) + \delta\mathbf{v}(\mathbf{r})$ . The associated changes in  $R_v$  and  $R_s$  are already given by equations (3.11) and (3.12), and those in  $\dot{F}$  are given by

$$\delta\dot{F} = \int d\mathbf{r}[-\mu\nabla\phi \cdot \delta\mathbf{v}] + \int dS[-L\partial_\tau\phi\delta v_\tau] = \int d\mathbf{r}[-\mu\partial_i\phi\delta v_i] + \int dS[-L\partial_\tau\phi\delta v_\tau]. \quad (3.33)$$

Combining equations (3.11), (3.12), (3.13), and (3.33), we obtain the Euler–Lagrange equations

$$-\eta\partial_j(\partial_j v_i + \partial_i v_j) - \frac{1}{2}\partial_i\alpha - \mu\partial_i\phi = 0 \quad (3.34)$$

in the bulk and

$$\eta(\partial_n v_\tau + \partial_\tau v_n) + \beta v_\tau^{\text{slip}} - L(\phi)\partial_\tau\phi = 0 \quad (3.35)$$

at the surface. Note that equation (3.34) is identical to the Stokes equation (3.31) with  $\partial_j v_j = 0$  and  $\alpha = -2p$ , and equation (3.35) reduces to the GNBC (3.32). An important point of this derivation is that the uncompensated Young stress at the boundary (last term on the left-hand side of equation (3.35)) must accompany the capillary force density in the bulk (last term on the left-hand side of equation (3.34)), both being the ‘elastic’ interfacial force. Hence the uncompensated Young stress is simply the manifestation of the fluid–fluid interfacial tension at the solid boundary.

Once the free energies  $\mathcal{F}[\phi]$  and  $\gamma_{fs}(\phi)$  are fixed, the contact-line motion (in the regime of small Reynolds number) is fully determined by equations (3.29), (3.30), (3.31), and (3.32), supplemented by the incompressibility condition  $\nabla \cdot \mathbf{v} = 0$  and the impermeability conditions  $v_n = 0$  and  $\partial_n \mu = 0$  at the solid surface (Qian *et al.* 2003). The Stokes equation can be readily generalized to the Navier–Stokes equation

$$\rho \left[ \frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla) \mathbf{v} \right] = -\nabla p + \eta \nabla^2 \mathbf{v} + \mu \nabla \phi, \quad (3.36)$$

by including the inertia forces, where  $\rho$  is the mass density. Together, the Navier–Stokes equation (3.36), the GNBC (3.32), the advection–diffusion equation (3.29), and equation (3.30) for the relaxation of interfacial  $\phi$ , form a consistent hydrodynamic model for the contact-line motion in immiscible two-phase flows, first presented by Qian *et al.* (2003). It is now clear that our model is necessitated by more general considerations.

#### 4. Comparison between MD and continuum results

To demonstrate the physical validity of our model, numerical solutions have been obtained for direct comparison to the MD velocity and interfacial profiles. For this purpose, we make use of the CH free energy functional (Cahn & Hilliard 1958)

$$\mathcal{F}_{CH}[\phi(\mathbf{r})] = \int d\mathbf{r} \left[ \frac{K}{2}(\nabla\phi)^2 + \left( -\frac{r}{2}\phi^2 + \frac{u}{4}\phi^4 \right) \right], \quad (4.1)$$

to fix the form of  $f(\phi)$  for  $\mathcal{F}[\phi]$  in equation (3.16). Here  $K$ ,  $r$ , and  $u$  are material parameters that can be determined from the interfacial thickness  $\xi = \sqrt{K/r}$ , the interfacial tension  $\gamma = 2\sqrt{2}r^2\xi/3u$ , and the two homogeneous equilibrium phases  $\phi_\pm = \pm\sqrt{r/u} = \pm 1$ .

The two coupled equations of motion are the advection–diffusion equation for the phase field  $\phi(\mathbf{r})$  and the Navier–Stokes equation in the presence of the capillary force

density:

$$\frac{\partial \phi}{\partial t} + \mathbf{v} \cdot \nabla \phi = M \nabla^2 \mu, \quad (4.2)$$

$$\rho \left[ \frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla) \mathbf{v} \right] = -\nabla p + \nabla \cdot \boldsymbol{\sigma}^v + \mu \nabla \phi + \mathbf{f}_e, \quad (4.3)$$

together with the incompressibility condition  $\nabla \cdot \mathbf{v} = 0$ . Here  $M$  is the mobility coefficient,  $\mu = \delta \mathcal{F}_{CH} / \delta \phi$  is the chemical potential derived from the CH free energy functional  $\mathcal{F}_{CH}$ ,  $\rho$  is the mass density of the fluid,  $p$  is the pressure,  $\boldsymbol{\sigma}^v = \eta[(\nabla \mathbf{v}) + (\nabla \mathbf{v})^T]$  is the Newtonian viscous stress tensor with  $\eta$  being the viscosity,  $\mu \nabla \phi$  is the capillary force density, and  $\mathbf{f}_e$  is the external force. (In a comparison between MD and continuum results, this  $\mathbf{f}_e$  is set to zero in the Couette flow geometry or to a constant parallel to the channel in the Poiseuille flow geometry. The continuum model is local, hence applicable to different flow geometries with different forms of  $\mathbf{f}_e$ .) The boundary conditions at the solid surface are the impermeability conditions  $\partial_n \mu = 0$ ,  $v_n = 0$ , the relaxational equation for the surface  $\phi$ :

$$\frac{\partial \phi}{\partial t} + v_\tau \partial_\tau \phi = -\Gamma L(\phi), \quad (4.4)$$

and the GNBC in continuum differential form:

$$\beta(\phi) v_\tau^{\text{slip}} = -\eta(\partial_n v_\tau + \partial_\tau v_n) + L(\phi) \partial_\tau \phi. \quad (4.5)$$

Here  $\tau$  denotes the direction tangent to the solid surface,  $n$  denotes the outward surface normal,  $\Gamma$  is a positive phenomenological parameter,  $L(\phi) = K \partial_n \phi + \partial \gamma_{fs}(\phi) / \partial \phi$  with  $\gamma_{fs}(\phi)$  being the fluid–solid interfacial free energy per unit area,  $\beta(\phi)$  is the slip coefficient which may locally depend on the local surface composition  $\phi$ , and  $L(\phi) \partial_\tau \phi$  is the uncompensated Young stress. We use  $\gamma_{fs}(\phi) = (\Delta \gamma_{fs} / 2) \sin(\pi \phi / 2)$  which is a smooth interpolation from  $\gamma_{fs}(\phi_-) = -\Delta \gamma_{fs} / 2$  to  $\gamma_{fs}(\phi_+) = \Delta \gamma_{fs} / 2$ . According to the Young equation for the static contact angle  $\theta_s$ :  $\gamma_{fs}(\phi_+) + \gamma \cos \theta_s = \gamma_{fs}(\phi_-)$ , we have  $\Delta \gamma_{fs} = -\gamma \cos \theta_s$ .

#### 4.1. Uncompensated Young stress

To interpret physically the second term on the right-hand side of equation (4.5), let us consider a fluid–fluid interface that intersects the planar solid surface  $z = 0$  with a contact angle  $\theta$  relative to the  $x$ -axis. For simplicity we assume the contact line to be straight and parallel to the  $y$ -axis, and hence  $\tau = x$ . If the fluid–fluid interface is gently curved, then  $\int_{\text{int}} dx [(K \partial_n \phi) \partial_x \phi] = \int_{\text{int}} d\phi (K \partial_m \phi) \cos \theta$ , where  $\int_{\text{int}} dx$  denotes the integration across the fluid–fluid interface along  $x$  and  $\partial_m$  is the spatial derivative along the fluid–fluid interface normal  $m$ , with  $\partial_n \phi \approx \partial_m \phi \cos \theta$ . As  $\int_{\text{int}} d\phi (K \partial_m \phi) = \gamma$ , we have  $\int_{\text{int}} dx [(K \partial_n \phi) \partial_x \phi] = \gamma \cos \theta$ . At equilibrium,  $L(\phi) = 0$ , and hence

$$\int_{\text{int}} dx [L(\phi) \partial_x \phi] = \int_{\text{int}} dx [(K \partial_n \phi + \partial \gamma_{fs} / \partial \phi) \partial_x \phi]$$

vanishes. This leads directly to the Young equation

$$\gamma \cos \theta_s + \gamma_{fs}(\phi_+) - \gamma_{fs}(\phi_-) = 0, \quad (4.6)$$

where  $\theta_s$  is the static contact angle,  $\gamma \cos \theta_s$  comes from  $\int_{\text{int}} dx [(K \partial_n \phi) \partial_x \phi]$ , and  $\gamma_{fs}(\phi_+) - \gamma_{fs}(\phi_-)$  from  $\int_{\text{int}} dx [(\partial \gamma_{fs} / \partial \phi) \partial_x \phi]$ . When the fluids are in motion, integrating the uncompensated Young stress  $L(\phi) \partial_x \phi$  across the fluid–fluid interface

along  $x$  yields

$$\int_{int} dx [L(\phi)\partial_x\phi] = \gamma \cos\theta_d + \gamma_{fs}(\phi_+) - \gamma_{fs}(\phi_-) = \gamma(\cos\theta_d - \cos\theta_s), \quad (4.7)$$

where  $\theta_d$  is the dynamic contact angle. Equation (4.7) implies that the uncompensated Young stress arises from the deviation of the fluid–fluid interface from its static configuration. However, it must be pointed out that here the contact angle is the so-called ‘microscopic contact angle’. It differs from the ‘apparent contact angle’ when the contact line is in motion. In particular, the apparent contact angle can change rather sharply when the contact line begins to move. But  $\theta_d$  can only vary linearly and relatively slowly with velocity.

From equation (4.7) and the fact that for moderate flow rates the field  $\phi$  for gently curved (fluid–fluid) interface relaxes to essentially the local stationary structure, we can write

$$L(\phi)\partial_x\phi = \gamma \cos\theta_d f(x) + \partial_x\gamma_{fs}, \quad (4.8)$$

where  $f(x)$  is a function which peaks at  $x_{CL}$ , the contact line centre, and  $\int dx f(x) = 1$ . In particular, when the static contact angle is  $90^\circ$  and  $\gamma_{fs}$  is a constant,  $f(x)$  can be approximated well by  $(3/4\sqrt{2}\xi)\cosh^{-4}[(x - x_{CL})/\sqrt{2}\xi]$ , derived from the stationary solution  $\phi_e = \phi_+ \tanh[(x - x_{CL})/\sqrt{2}\xi]$ . Thus the uncompensated Young stress is a peaked function centred at the contact line (note that  $\partial_x\gamma_{fs}$  is also peaked at the interfacial region). While the above expressions give a physical interpretation of the uncompensated Young stress, it is noted that the presence of  $\cos\theta_d$  makes the expression awkward for the purpose of calculation (as  $\theta_d$  should be the outcome of the calculation). Thus in computations the formulation in terms of  $\phi$  is preferred.

#### 4.2. Parameter considerations

We have carried out the MD–continuum comparison in such a way that adjustable parameters are hardly involved in the model calculation (Qian *et al.* 2003). This is achieved as follows. There is a total of nine material parameters in our model:  $\rho$ ,  $\eta$ ,  $\beta$ ,  $\xi$ ,  $\gamma$ ,  $|\phi_\pm|$ ,  $\theta_s$ ,  $M$ , and  $\Gamma$ . Among these parameters, the first seven are directly measurable in MD simulations according to their respective physical definitions. For example,  $\eta$  is the viscosity which is obtained from the ratio of the shear viscous stress to the shear rate, both directly measurable in a steady-state flow,  $\gamma$  is the interfacial tension which is an integrated measure of the microscopic stress anisotropy across the fluid–fluid interface, and  $\theta_s$  is the static contact angle which is directly measurable in the static configuration. The values of  $M$  and  $\Gamma$  are fixed through an optimized MD–continuum comparison: one flow field from an MD simulation is best fitted/matched by that from a hydrodynamic model calculation with optimized  $M$  and  $\Gamma$  values (selected from a scan of the parameter values), although in our case the fitting is not very sensitive to the values of  $M$  and  $\Gamma$ . That is, fitting can be almost as good if  $M$  and  $\Gamma$  deviate from the optimal values. It will be shown in §5 that such insensitivity is not an accident, i.e. as long as the values of  $M$  and  $\Gamma$  are in the right range for the hydrodynamic model to reach the sharp interface limit, the continuum predictions should not be sensitive to these parameter values.

#### 4.3. Couette flow and Poiseuille flow comparisons

Once all the parameter values are determined, predictions from our hydrodynamic model can be readily compared to the results from a series of MD simulations with different external conditions. The overall agreement is excellent, thus demonstrating the validity of the GNBC and the hydrodynamic model. We emphasize that the

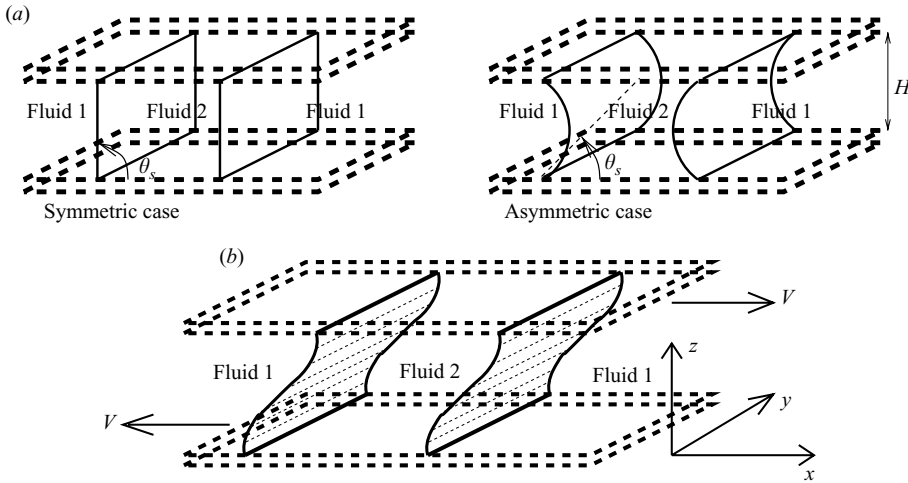


FIGURE 2. Schematic of simulation geometry. (a) Static configurations in the symmetric and asymmetric cases. Fluid 2 is sandwiched by fluid 1 due to the periodic boundary condition along the  $x$ -direction. (b) Dynamic configuration in the symmetric case.

MD–continuum agreement has been achieved from the molecular-scale vicinity of the contact line (Qian *et al.* 2003) to far fields at the large scale (Qian *et al.* 2004). The numerical algorithm for solving the continuum model is a second-order scheme using a modified version of the pressure-Poisson formulation (Johnston & Liu 2002), where the incompressibility condition is replaced by the pressure Poisson equation and a divergence-free boundary condition. For details, see Appendix C of Qian *et al.* (2003).

MD simulations have been carried out for immiscible two-phase flows in the Couette geometry (see figure 2) (Qian *et al.* 2003). Two immiscible fluids were confined between two planar solid walls parallel to the  $(x, y)$ -plane, with the fluid–solid interfaces defined at  $z=0$  and  $H$ . The Couette flow was generated by moving the top and bottom walls at a constant speed  $V$  in the  $\pm x$ -directions, respectively. Periodic boundary conditions were imposed along the  $x$ - and  $y$ -directions. Technical details of our MD simulations may be found in Qian *et al.* (2003, 2004). Two cases were considered. In the symmetric case, the static contact angle  $\theta_s$  is  $90^\circ$  and the fluid–fluid interface is flat, parallel to the  $(y, z)$ -plane. In the asymmetric case, the static contact angle  $\theta_s$  is  $64^\circ$ , and the fluid–fluid interface is curved in the  $(x, z)$ -plane. Steady-state velocity and interfacial profiles were obtained from time averages over  $10^5 \tau$  or longer, where  $\tau$  is the atomic time scale  $\sqrt{m\sigma^2/\epsilon}$ , with  $\epsilon$  and  $\sigma$  being the energy and length scales in the Lennard–Jones potential for fluid molecules, and  $m$  the fluid molecular mass. Throughout the remainder of this paper, all physical quantities are given in terms of the Lennard–Jones reduced units (defined in terms of  $\epsilon$ ,  $\sigma$ , and  $m$ ).

Figure 3 shows the MD and continuum velocity fields for a symmetric case of Couette flow, figure 4 shows those fields for an asymmetric case of Couette flow, and figure 5 shows the MD and continuum fluid–fluid interface profiles for the above two cases, from which the difference between the ‘apparent’ advancing and receding contact angles is clearly seen.

In order to further verify that the hydrodynamic model is local and the parameter values are local properties, hence applicable to different flow geometries, we have

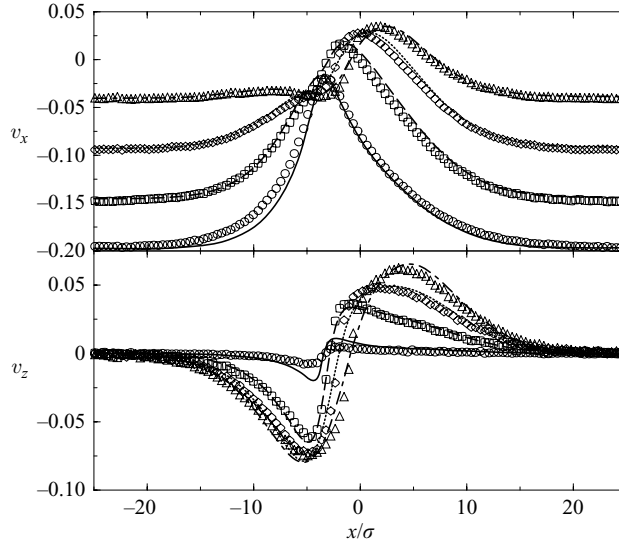


FIGURE 3. Comparison of the MD (symbols) and continuum (lines) velocity profiles ( $v_x(x)$  and  $v_z(x)$  at different  $z$  levels) for a symmetric case of immiscible Couette flow ( $V = 0.25(\epsilon/m)^{1/2}$  and  $H = 13.6\sigma$ ). The profiles are symmetric about the centreplane  $z = H/2$ , hence only the lower half is shown at  $z = 0.425\sigma$  (circles and solid lines),  $2.125\sigma$  (squares and dashed lines),  $3.825\sigma$  (diamonds and dotted line), and  $5.525\sigma$  (triangles and dot-dashed lines).

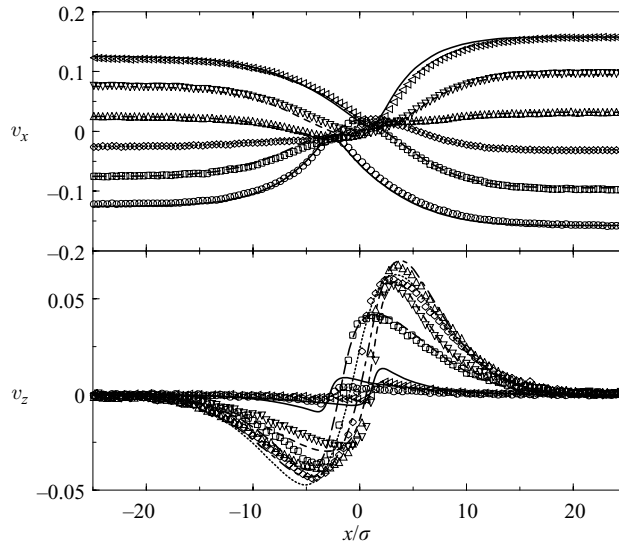


FIGURE 4. Comparison of the MD (symbols) and continuum (lines) velocity profiles ( $v_x(x)$  and  $v_z(x)$  at different  $z$  levels) for an asymmetric case of immiscible Couette flow ( $V = 0.2(\epsilon/m)^{1/2}$  and  $H = 13.6\sigma$ ), shown at  $z = 0.425\sigma$  (circles and solid lines),  $2.975\sigma$  (squares and long-dashed lines),  $5.525\sigma$  (diamonds and dotted line),  $8.075\sigma$  (up-triangles and dot-dashed lines),  $10.625\sigma$  (down-triangles and dashed lines),  $13.175\sigma$  (left-triangles and solid lines). Although the solid lines are used to denote two different  $z$  levels, for each solid line, whether it should be compared to circles or left-triangles is self-evident.

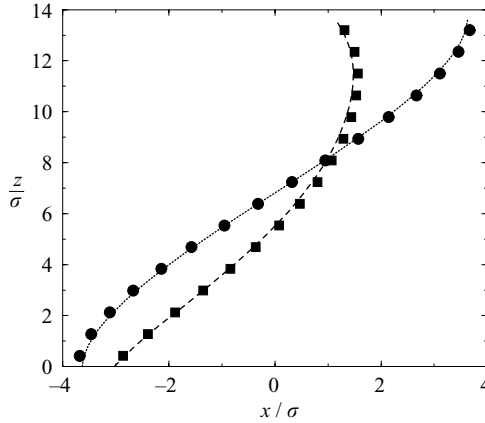


FIGURE 5. Comparison of the MD (symbols) and continuum (lines) fluid–fluid interface profiles, defined by  $\rho_1 = \rho_2$  ( $\phi = 0$ ). The circles and dotted line denote the symmetric immiscible Couette flow with  $V = 0.25(\epsilon/m)^{1/2}$  and  $H = 13.6\sigma$ ; the squares and dashed line denote the asymmetric immiscible Couette flow with  $V = 0.2(\epsilon/m)^{1/2}$  and  $H = 13.6\sigma$ .

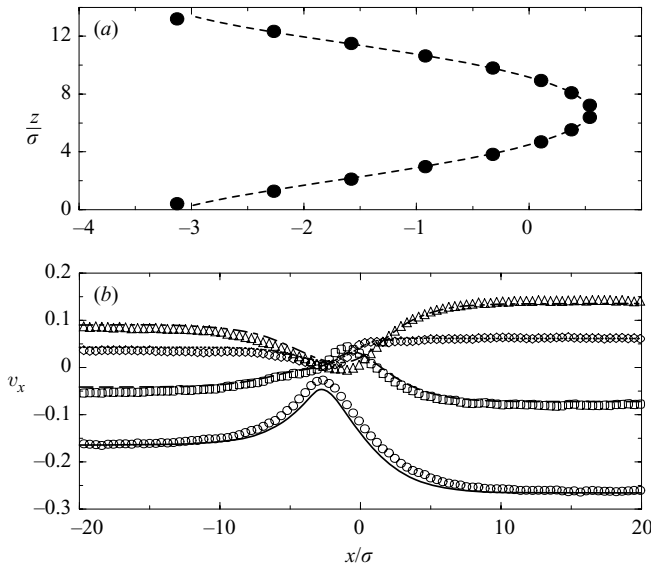


FIGURE 6. Comparison of the MD (symbols) and continuum (lines) results for an asymmetric case of immiscible Poiseuille flow. An external force  $mg_{ext} = 0.05\epsilon/\sigma$  is applied on each fluid molecule in the  $x$ -direction, and the two walls, separated by  $H = 13.6\sigma$ , move at a constant speed  $V = 0.51(\epsilon/m)^{1/2}$  in the  $-x$ -direction to maintain a stationary steady-state interface. (a) Fluid–fluid interface profiles, defined by  $\rho_1 = \rho_2$  ( $\phi = 0$ ). (b)  $v_x(x)$  at different  $z$  levels. The profiles are symmetric about the centreplane  $z = H/2$ , hence only the lower half is shown at  $z = 0.425\sigma$  (circles and solid line),  $2.125\sigma$  (squares and dashed line),  $3.825\sigma$  (diamonds and dotted line), and  $5.525\sigma$  (triangles and dot-dashed line).

carried out MD and continuum simulations for immiscible Poiseuille flows (Qian *et al.* 2003). We find that the hydrodynamic model with the same set of parameters is capable of reproducing the MD velocity and interfacial profiles, shown in figure 6. Similar to what is observed in Couette flows, here the slip is near-complete at the MCL, i.e.  $v_x \approx 0$  and  $|v_x^{slip}| \approx V$  (the wall speed), while far away from the contact

line, the flow field is not perturbed by the fluid–fluid interface and the single-fluid unidirectional Poiseuille flow is recovered.

The parameter values used for the MD–continuum comparison have been given by Qian *et al.* (2003). We emphasize that the overall agreement is excellent in all cases, therefore the validity of the GNBC and the hydrodynamic model is well-affirmed.

It should be mentioned that our MD results also show some fluid–fluid interfacial structure which is outside the realm of our continuum model. In particular, the MD data show a fairly significant density drop at the interface ( $\phi = 0$ ). Physically this is due to the repulsive molecular interaction between the two immiscible fluids. This density drop has implications for the slip coefficient  $\beta$  in that in the interfacial region  $\beta$  is no longer a simple composition of  $\beta_1$  and  $\beta_2$  on the two sides of the contact line. At the same time, MD data also revealed and verified the form and nature of the uncompensated Young stress, as predicted by equations (4.7) and (4.8). Details can be found in Qian *et al.* (2003).

#### 4.4. Power-law decay of partial slip

Another important comparison between MD and continuum hydrodynamics is the behaviour that interpolates between the MCL, where there is near-complete slip, and far away from the contact line, where there is at most a small amount of partial slip.

MD simulations have been carried out for immiscible Couette flows in increasingly wider channels (Qian *et al.* 2004). The inset to figure 7 shows the tangential velocity profiles at the wall. Immediately next to the MCL, there is a small core region, where the slip profiles show a sharp decay within a few  $l_s$  (the slip length in single-phase flows). As the channel width  $H$  increases, a much more gentle variation of the slip profiles becomes apparent. In order to reveal the nature of this slow variation, we plot in figure 7 the same data in the log-log scale. The dashed line has a slope  $-1$ , indicating the  $1/x$  behaviour of the slip profile, where  $x$  is the distance from the MCL. Because of the finite  $H$ , there is always a plateau in each of the single-phase flow regions, where the constant small amount of slip is given by  $v_0^{slip} = 2Vl_s/(H + 2l_s)$ , which acts as an outer cutoff on the  $1/x$  profile. This expression for  $v_0^{slip}$  is simply derived from the NBC and the Navier–Stokes equation for uniform shear flow. Our largest MD simulation shows that the  $v^{slip} \propto 1/x$  behaviour extends to  $\sim 50\sigma$  (or  $\sim 25l_s$ ). Obviously, as  $H \rightarrow \infty$  and  $v_0^{slip}$  approaches 0 (no slip), the power-law region can be very wide indeed. This large  $1/x$  partial-slip region indicates that the outer cutoff length scale (e.g. the system dimension) would determine the integrated effects, such as the total steady-state dissipation. In the past the similarity solutions of the Stokes equation have shown the  $1/x$  stress variation away from the MCL (Moffatt 1964; Huh & Scriven 1971). However, to our knowledge the fact that the partial slip also exhibits the same spatial dependence was first determined by Qian *et al.* (2004), even though the validity of the NBC has been verified at high shear stress (Thompson & Troian 1997; Barrat & Bocquet 1999a; Cieplak *et al.* 2001).

The continuum results shown in figure 7 were obtained on a uniform mesh, using the same set of parameter values corresponding to the same local properties in all the five MD simulations. We have extended the MCL simulations, through continuum hydrodynamics, to lower flow rates and much larger systems. For this purpose, we have employed the adaptive method based on iterative grid redistribution (Ren & Wang 2000). The computational mesh is redistributed following the behaviour of the continuum solution so that there is fine molecular resolution in the interfacial region, while elsewhere a much coarser mesh is used to save computational cost.



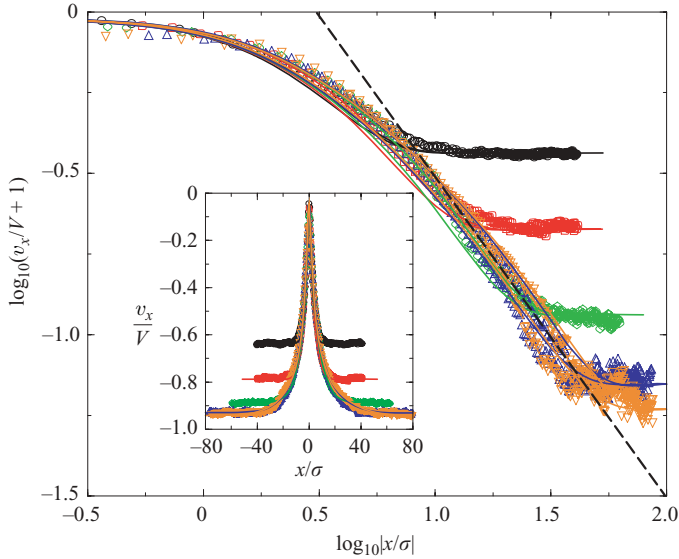


FIGURE 7. Log-log plot for the slip profiles. Here  $v_x/V+1$  is the scaled slip velocity at the lower fluid–solid interface  $z=0$ , and  $x/\sigma$  measures the distance from the MCL in units of  $\sigma$ . The wall is moving at  $-V$ , hence  $v_x/V=0$  means complete slip and  $v_x/V=-1$  means no slip. The  $v_x$  profiles were obtained for five symmetric cases of Couette flow, with different values for  $H$  but the same value for  $V$  ( $=0.05\sqrt{\epsilon/m}$ ) and also the same parameters for densities and interactions. The symbols represent the MD results and the solid lines represent the continuum results, obtained for  $H=6.8\sigma$  (black circles and line),  $H=13.6\sigma$  (red squares and line),  $H=27.2\sigma$  (green diamonds and line),  $H=54.4\sigma$  (blue up-triangles and line),  $H=68\sigma$  (orange down-triangles and line). There are two solid curves for each colour, one for the slip profile to the left of the MCL and the other to the right of the MCL. The dashed line has the slope of  $-1$ , indicating that the  $1/x$  behaviour is approached for increasingly larger  $H$ . For  $H=68\sigma$ , the  $1/x$  behaviour extends from  $|x|\approx 12\sigma\approx 6l_s$  to  $50\sigma\approx 25l_s$ , where  $l_s$  was measured to be  $2\sigma$ . Inset: The scaled tangential velocity  $v_x/V$  at  $z=0$ , plotted as a function of  $x/\sigma$ .

A semi-implicit time-stepping scheme is also used to speed up the approach to steady state.

The continuum results for three large systems of low flow rates are shown in figure 8. The capillary force is verified to be important in the interfacial region only, whereas the pressure gradients and viscous forces show a much slower variation. They are balanced outside the interfacial region, and hence the flow is governed by the Stokes equation. This is expected, because the Reynolds number  $m\rho VH/\eta\approx 0.6$  for  $\rho\approx 0.8/\sigma^3$ ,  $V=0.005\sqrt{\epsilon/m}$ ,  $H\approx 300\sigma$ , and  $\eta\approx 2.0\sqrt{\epsilon m}/\sigma^2$ . Figure 8 shows the slip profiles plotted on the log-log scale, which clearly exhibit the  $1/x$  behaviour extending from  $|x|\approx 6l_s$  to  $|x|\approx 270l_s$ . The inset to figure 8 shows the scaled tangential velocity profiles at the solid surface, from which the existence of a universal slip profile is evident. Physically, when  $H\gg l_s$ , the Stokes flow is governed by only one velocity scale  $V$  and one length scale  $l_s$ . Thus universality becomes evident from  $v_x/V$  plotted as a function of  $x/l_s$ . A heuristic account of the universal slip profile is as follows. Away from the MCL, the viscous shear stress is given by  $-a\eta v_x(x)/|x|$ , where  $a$  is a constant  $\sim 1$  and  $v_x(x)$  is the local tangential velocity. The NBC implies  $v_x^{slip}(x)=-al_s v_x(x)/|x|$ . Combining this equation with  $v_x^{slip}=v_x+V$  yields  $v_x^{slip}(x)/V=1/(1+|x|/al_s)$ . This

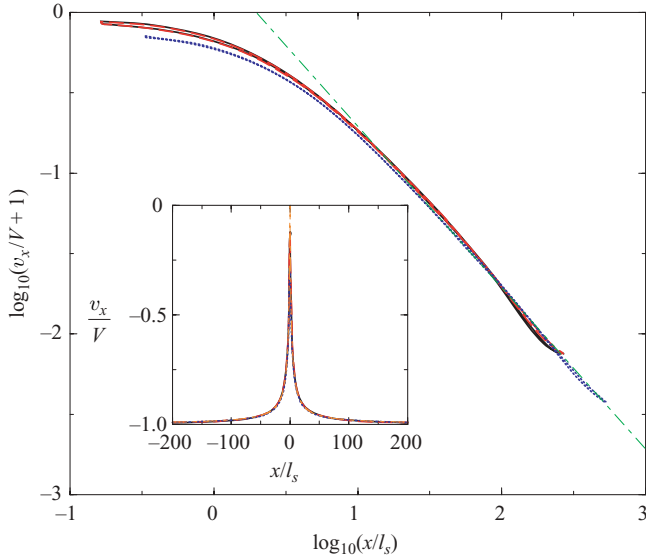


FIGURE 8. Log-log plot of the slip profiles. Here  $v_x/V+1$  is the scaled slip velocity at the lower fluid–solid interface  $z=0$ , and  $x/l_s$  measures the distance from the MCL in units of  $l_s$ . The wall is moving at  $-V$ , hence  $v_x/V=0$  means complete slip and  $v_x/V=-1$  means no slip. There are three symmetric cases of Couette flow. The black solid line denotes the case of  $H=326\sigma$ ,  $V=0.005\sqrt{\epsilon/m}$  and  $l_s=1.24\sigma$ , the red dashed line denotes the case of  $H=326\sigma$ ,  $V=0.0025\sqrt{\epsilon/m}$  and  $l_s=1.24\sigma$ , and the blue dotted line denotes the case of  $H=326\sigma$ ,  $V=0.0025\sqrt{\epsilon/m}$  and  $l_s=0.62\sigma$ . The green dot-dashed line has the slope of  $-1$ , indicating a  $1/x$  region much wider than that in figure 7. Inset: Universal slip profile. The scaled tangential velocity  $v_x/V$  at  $z=0$  is plotted as a function of the scaled coordinate  $x/l_s$  for the three cases. The slip profiles exhibit a partial-slip region hundreds of  $l_s$  in size. The relation  $v_x/V=1/(1+|x|/2.14l_s)-1$  is also plotted by the orange dot-dashed line, showing an extremely good fit.

relation, with  $a \approx 2.14$  for best fit, agrees with the continuum slip profiles extremely well, as seen in the inset to figure 8.

## 5. Scaling analysis: sharp or diffusive interface limit

In this section we look at the MCL problem from two different perspectives. These naturally arise from the physical reality that there are two distinct regions in the MCL problem: the interfacial region and the rest. In the perspective of the sharp interface limit, the problem is looked at from outside the interfacial region. The effect of the interfacial region is taken into account only in an integrated sense. That is, the interface is impenetrable, the interfacial thickness is much smaller than any characteristic hydrodynamic length scale, and the mechanical effect of the interface is manifested through the interfacial tension and curvature only. In the diffusive interface limit, on the other hand, we examine the problem when the interfacial region is physically large. It turns out that the two limits can be mathematically analysed by varying the magnitude of  $M$  and  $\Gamma$ . In addition, we also give an abbreviated account of the recent work by Ren & E (2005b) on the sharp interface limit, which not only simplifies the mathematics, but also expresses the GNBC without the  $\phi$  variable, and hence is physically more transparent.

In Appendix B we have derived the total rate of energy dissipation in steady state (see equations (B 3), (B 4), and (B 5)),

$$\begin{aligned}
 R_2[\mathbf{v}] &= R_v[\mathbf{v}] + R_s[\mathbf{v}] + R_d[\mathbf{v}] + R_r[\mathbf{v}] \\
 &= \int d\mathbf{r} \left[ \frac{\eta}{2} (\partial_i v_j + \partial_j v_i)^2 \right] + \int dS [\beta (v_\tau^{\text{slip}})^2] \\
 &\quad + \int d\mathbf{r}_1 d\mathbf{r}_2 \left[ -\frac{1}{M} G(\mathbf{r}_1, \mathbf{r}_2) (\mathbf{v} \cdot \nabla \phi)_{r_1} (\mathbf{v} \cdot \nabla \phi)_{r_2} \right] + \int dS \left[ \frac{1}{\Gamma} (v_\tau \partial_\tau \phi)^2 \right], \quad (5.1)
 \end{aligned}$$

where  $G(\mathbf{r}_1, \mathbf{r}_2)$  is the Green function for the Laplacian operator satisfying the boundary condition  $\partial_n \mu = 0$  at the solid surface. This function(al) is useful for the variational analysis of the sharp/diffuse interface limits. It is obtained by eliminating the two rates  $(\mathbf{J}, \dot{\phi})$  by expressing them in terms of  $\mathbf{v}$ , for the steady state. The Stokes equation (3.31) and the GNBC (3.32) can be derived by minimizing  $R_2[\mathbf{v}]$  with respect to  $\mathbf{v}$ , supplemented with equations (3.29) and (3.30) for the linear dissipative dynamics of  $\phi$  (see equations (B 8) and (B 9)).

### 5.1. Sharp interface limiting behaviours

The CH diffuse-interface modelling allows diffusive transport through the fluid–fluid interface (Seppecher 1996; Jacqmin 2000; Chen *et al.* 2000; Pismen & Pomeau 2000; Briant & Yeomans 2004). By taking the limit of  $M \rightarrow 0$  and  $\Gamma \rightarrow 0$ , we obtain the sharp interface limit in which advection dominates such that  $\dot{\phi} \rightarrow 0$  and  $\partial\phi/\partial t \rightarrow -\mathbf{v} \cdot \nabla\phi$ . According to equation (5.1), if  $1/M$  and  $1/\Gamma$  both approach positive infinity, minimizing  $R_2$  with respect to  $\mathbf{v}$  then requires  $\mathbf{v} \cdot \nabla\phi \rightarrow 0$  in the interfacial region (see the last two terms on the right-hand side of equation (5.1)). That means in steady state the flow is parallel to the interface.

The limiting magnitude of  $\mathbf{v} \cdot \nabla\phi$  can be obtained through a scaling analysis. We assume that the interfacial thickness  $\xi$  is much smaller than the smallest interfacial curvature radius, a limit realized for moderate shear rates. Integrating the Stokes equation (3.31) across the fluid–fluid interface yields  $|\mu|\phi_+ \sim \eta\|\nabla\mathbf{v}\|$ , from which we have  $|\mu| \sim \eta V/l\phi_+$ , where  $V$  and  $l$  are the characteristic velocity and length scales of the flow. Integrating equation (3.29) (with  $\partial\phi/\partial t = 0$  in steady state) across the fluid–fluid interface yields  $|v_m|\phi_+ \sim M\|\nabla\mu\|$ , from which we have  $|v_m| \sim M|\mu|/l\phi_+$ , where  $v_m$  is the velocity component along the interfacial normal  $m$ . Together,  $|\mu| \sim \eta V/l\phi_+$  and  $|v_m| \sim M|\mu|/l\phi_+$  lead to  $|v_m|/V \sim M\eta/l^2\phi_+^2$  for the dimensionless interfacial normal velocity. This relation indicates that in order to realize the sharp interface limit,  $\sqrt{M\eta}/\phi_+$  is the length scale that must be made small enough compared to  $l$ . (This length scale arises from the coupling of equations (3.31) and (3.29) (Bray 1994; Jacqmin 2000).) The characteristic length  $l$  of the flow field is given by the slip length  $l_s$  in the inner region close to the contact line, where  $l_s$  is in fact the only length scale governing the Stokes flow. Far away from the contact line,  $l$  becomes the dimension of the confined system, e.g.  $l \sim H$ . Typically  $l_s$  is much smaller than  $H$ , and consequently  $|v_m|$  reaches the maximum in the immediate vicinity of the MCL. This leads to the first conclusion, that the sharp interface limit is realized in the bulk when

$$\frac{|v_m|}{V} \sim \frac{M\eta}{l_s^2\phi_+^2} \ll 1. \quad (5.2)$$

This relation ensures that the flow is parallel to the interface in the inner region of fast velocity variation and large interfacial curvature.

In the sharp interface limit, the tangential viscous stress is vanishingly small at the MCL because of the vanishing interfacial normal velocity there. Here we consider the case of  $\theta_s = 90^\circ$  for simplicity and generalization to  $\theta_s \neq 90^\circ$  is straightforward. It follows that in steady state the near-complete slip at the MCL is mainly sustained by the uncompensated Young stress:  $\beta V \sim \gamma \cos \theta_d / \xi$ , where  $V$  is the maximum slip velocity (the wall speed relative to the stationary fluid–fluid interface) and  $\gamma \cos \theta_d / \xi$  is obtained by considering the integrated uncompensated Young stress in equation (4.7) as distributed within  $\xi$ . The difference between the dynamic and static contact angles,  $\Delta\theta = \theta_s - \theta_d$ , is then obtained as  $\Delta\theta \sim \xi \beta V / \gamma = \xi Ca / l_s$ , where  $l_s = \eta / \beta$  is the slip length and  $Ca = \eta V / \gamma$  is the capillary number. Integrating equation (3.30) (with  $\partial\phi/\partial t = 0$ ) across the fluid–fluid interface along the solid surface yields the tangential velocity across the interface  $|v_\tau| \sim \Gamma K \cos \theta_d \approx \Gamma K \Delta\theta$ . (For  $\theta_s = 90^\circ$ , the tangential direction  $\tau$  is the same as the interfacial normal  $m$ , and hence  $v_\tau$  becomes  $v_m$ .) Substituting  $\Delta\theta \sim \xi Ca / l_s$  into  $|v_m| \sim \Gamma K \Delta\theta$ , we obtain  $|v_m| \sim \Gamma K \xi Ca / l_s$ , and hence  $|v_m| / V \sim \Gamma K \xi \eta / \gamma l_s$ . Using  $\gamma = 2\sqrt{2}K\phi_+^2/3\xi$ , we obtain  $|v_m| / V \sim \Gamma \xi^2 \eta / \phi_+^2 l_s$ . This leads to the second conclusion, that the sharp interface limit is realized at the contact line when

$$\frac{|v_m|}{V} \sim \frac{\Gamma \xi^2 \eta}{\phi_+^2 l_s} \ll 1, \quad (5.3)$$

which ensures that the interface is not penetrated by the flow. Note that the two conditions (5.2) and (5.3) are independent of the magnitude of  $V$ . This is consistent with the linearity of the hydrodynamic model (for sufficiently low flow rates) and has been verified well numerically.

In §4, it has been mentioned that  $M$  and  $\Gamma$  are treated as fitting parameters to optimize the MD–continuum comparison. Since the fluid–fluid interface in our MD simulations is impenetrable, the hydrodynamic model has to be within the sharp interface limit in order to reproduce the MD results. It follows that the values of  $M$  and  $\Gamma$  must satisfy the conditions (5.2) and (5.3). Moreover, as long as the sharp interface limit is reached, the continuum predictions are not sensitive to the values of  $M$  and  $\Gamma$ . Such has indeed been our experience. Thus  $M$  and  $\Gamma$  should not be regarded as fitting parameters in our MD–continuum comparison because they are simply used to realize the sharp interface limit, in accordance with the interface impenetrability conditions (viewed outside the interfacial region).

We have shown that in order to sustain the near-complete slip at the MCL, the dynamic contact angle  $\theta_d$  must deviate from the static angle  $\theta_s$  by  $\Delta\theta \sim \xi Ca / l_s$ . A similar scaling relation has recently been obtained by Ren & E (2005b) in a general discussion on the sharp interface limit of the GNBC. Here we derive a complete expression for  $\Delta\theta$  within the CH phase-field formulation. Consider an interface deformed by the shearing movement of confining walls (see figure 2b). In steady state the tangential viscous stress is negligibly small at the MCL where the slip is near complete ( $v_x^{slip} \approx V$  at the lower fluid–solid interface). Therefore the GNBC may be expressed as  $\beta V = -K \partial_z \phi \partial_x \phi$ . Suppose the deviation of  $\theta_d$  from  $\theta_s = 90^\circ$  is very small (this is indeed the case, as we will show below). Then  $\partial_z \phi \approx -\partial_x \phi \cos \theta_d$  and  $\beta V \approx K (\partial_x \phi)^2 \cos \theta_d$ . Using the interfacial profile  $\phi(x) = \phi_+ \tanh(x/\sqrt{2}\xi)$  at  $z = 0$  (for a gently deformed interface), we obtain  $\beta V \approx K \phi_+^2 \cos \theta_d / 2\xi^2$ . It follows that

$$\Delta\theta = \frac{4\sqrt{2}\xi}{3} \frac{\eta V}{l_s \gamma}. \quad (5.4)$$

This expression has been quantitatively verified. Substituting the MD values  $l_s = 3.8\xi$ ,  $\gamma = 5.5\epsilon/\sigma^2$ ,  $\eta = 1.95\sqrt{\epsilon m}/\sigma^2$ , and  $V = 0.25\sqrt{\epsilon/m}$  into equation (5.4) yields  $\Delta\theta = 0.044$  (or  $2.5^\circ$ ), in excellent agreement with  $\cos\theta_d = 0.0437$  obtained from the  $\phi = 0$  locus in the continuum solution. There is another measure of  $\cos\theta_d$  using the integrated Young stress  $\int_{int} dx K \partial_n \phi \partial_x \phi = \gamma \cos\theta_d$ , which produces a value only slightly different from that determined by the  $\phi = 0$  locus. We note that in our nanoscale MD simulations,  $l_s$  is typically larger than  $\xi$  and  $\gamma > 10\eta V$ , and hence  $\Delta\theta < 0.1$ . Yet this small angle of deviation is necessary for the uncompensated Young stress to sustain the near-complete slip of the MCL (Qian *et al.* 2003).

In a recent study of the sharp interface limit of the GNBC by Ren & E (2005*b*), it has been shown that the deviation of the dynamic contact angle  $\theta_d$  from the static contact angle  $\theta_s$  is proportional to the dimensionless parameter  $\beta^* V \delta / \gamma$ , which measures the relative strength of the frictional force between the fluid and the solid and the interfacial force between the fluids. Here  $\beta^*$  is the (average) slip coefficient in the contact line region, depending on  $\beta_1$  and  $\beta_2$  in the two single-phase flow regions as well as on the fluid–fluid interfacial structure, and  $\delta$  is the fluid–fluid interfacial thickness. Numerical results have been obtained for the relation between the microscopic contact angle  $\theta_d$  and the apparent contact angle, demonstrating good agreement with the analytical results based on matched asymptotic expansions (Cox 1986).

From the above, it follows that in the sharp interface limit the contact angle can be set at the value of the static contact angle (for  $\xi \rightarrow 0$  and/or  $Ca \rightarrow 0$ ), and since outside the fluid–fluid interfacial region the NBC is valid, numerically the flow field can be calculated separately on the two sides of the interface, linked together via the interface transition relations (zero normal component of velocity, continuity of tangential component of velocity, continuity of tangential stress, normal stress difference across the interface being balanced by the tensile force proportional to the interface curvature) (Zhou & Sheng 1990; Ren & E 2005*b*). However, in such numerical solutions the tangential (viscous) stress is necessarily discontinuous at the contact line. This can be clearly seen by considering the tangential viscous stresses approached along the three interfaces (two fluid–solid and one fluid–fluid) terminating at the contact line. For simplicity, let us consider a fluid–fluid interface vertically intersecting the solid surface. Velocity continuity dictates that the tangential viscous stress at the solid surface approaches  $\beta_1 V$  (or  $\beta_2 V$ ) at the contact line in the single-phase flow region left (or right) of the fluid–fluid interface, with  $\beta_1$  and  $\beta_2$  being the slip coefficients in the left and right regions, respectively. However, the impenetrability condition at the fluid–fluid interface dictates that along this interface, the velocity component parallel to the solid surface vanishes, leading to a vanishing tangential viscous stress at the solid surface when approached along the fluid–fluid interface down to the contact line. Hence there exist three distinct values for the tangential viscous stress. The uncompensated Young stress thus enters as the required subsidiary condition to complete the picture and make the solutions physically meaningful. In particular, the uncompensated Young stress interpolates between the two values of  $\beta_1 V$  and  $\beta_2 V$ , with a mean value given by  $(\beta_1 + \beta_2)V/2$ , if the two fluids are assumed to interact with the solid independently (Qian *et al.* 2003).

### 5.2. Diffusive interface limit

Opposite to the sharp interface limit is the limit of  $M \rightarrow \infty$  and  $\Gamma \rightarrow \infty$ . In this limit minimizing  $R_2$  in equation (5.1) is equivalent to minimizing  $R_1$  in equation (3.8) because  $R_d$  and  $R_r$  both vanish regardless of the velocity distribution (see equations

(B3) and (B4)). As a consequence, the flow field and the fluid–fluid interface are decoupled: the velocity is distributed as if there were only one single phase while the interfacial profile approaches the equilibrium one. As the fluid–fluid interface becomes very transparent (through diffusive transport), the contact line loses its usual implications. Indeed, it has been shown that as this limit is approached, the stress singularity can be removed even if the no-slip boundary condition is applied (Seppecher 1996; Jacqmin 2000; Chen *et al.* 2000; Briant & Yeomans 2004). This result can be made intuitively plausible from our variational formulation as follows.

According to equation (5.1), if the functional  $R_2[\mathbf{v}]$  is to be minimized subject to  $\beta$ ,  $1/M$ , and  $1/\Gamma$  all approaching positive infinity, then the problem is reduced to solving the Stokes equation subject to the no-slip boundary condition  $v_\tau^{slip} = 0$  and the interface impenetrability condition  $v_m = 0$ . This leads to the well-known non-integrable singularity in viscous dissipation. Therefore, mathematically the contact-line singularity may be viewed as resulting from minimizing  $R_2[\mathbf{v}]$  with  $\beta$ ,  $1/M$ , and  $1/\Gamma \rightarrow \infty$ . By removing either the  $\beta \rightarrow \infty$  constraint (i.e. allowing slipping), or the  $1/M, 1/\Gamma \rightarrow \infty$  constraint (i.e. allowing diffusive relaxation), the total dissipation can only decrease from infinity, thus regularizing the solution. This is especially the case since the divergence is logarithmic in nature, i.e. the divergence is only marginal. We should note, however, that physically realistic cases correspond to  $\beta$ ,  $1/M$ , and  $1/\Gamma$  all remaining finite, as evidenced by MD results. In fact, application of the above considerations to the problem of corner-flow singularity (involving a flow in a corner with one rigid plane sliding over another) (Batchelor 1967; Moffatt 1964; Koplik & Banavar 1995) would be equally valid (Qian & Wang 2005).

### 5.3. Interfacial dissipation

It is interesting to note that in either of the two limits discussed above, the rate of interfacial dissipation,  $R_d + R_r$ , tends to vanish. In the sharp interface limit of  $M \rightarrow 0$  and  $\Gamma \rightarrow 0$ , the limiting behaviours of the interfacial normal velocity expressed in equations (5.2) and (5.3) give  $R_d \sim M$  and  $R_r \sim \Gamma$ . (Equation (B3) indicates  $R_d \propto |v_m|^2/M$  while equation (5.2) indicates  $|v_m| \propto M$ , and hence  $R_d \propto M \rightarrow 0$ . Similarly,  $R_r \propto \Gamma \rightarrow 0$ .) In the opposite limit of  $M \rightarrow \infty$  and  $\Gamma \rightarrow \infty$ , as the interface is penetrated by the flow,  $R_d$  and  $R_r$  simply vanish proportionally to  $1/M$  and  $1/\Gamma$ , respectively. That the positive definite rate of interfacial dissipation,  $R_d + R_r$ , approaches zero in the two opposite limits implies that a maximum should be reached somewhere in between. However, the total rate of dissipation  $R_2$  should increase monotonically from the limit of  $M \rightarrow \infty$  and  $\Gamma \rightarrow \infty$  to that of  $M \rightarrow 0$  and  $\Gamma \rightarrow 0$ . Although there is only  $R_v + R_s$  left in  $R_2$  in either of the two limits, the latter (sharp interface) limit imposes vanishing interfacial normal velocity as the additional condition. This would certainly lead to a flow whose total rate of dissipation is larger than that obtained from minimizing the same functional without the additional constraint.

Note that while mathematically the sharp interface limit may be simply obtained by excluding  $R_d$  and  $R_r$  from  $R_2$  and applying  $v_m = 0$  at the interface instead, physically the two-phase interfacial dissipation may not always be negligible, especially when the interfacial region of partial miscibility has finite and non-negligible width such that structural relaxation may occur.

It should be noted that dimensional analysis indicates that  $[M] = [\Gamma] [\text{Length}]^3$ , i.e. there is a length scale  $l_0$  which links these two parameters through the relation  $M = \Gamma l_0^3$ . Physically it is plausible to assume that  $l_0$  is determined by a combination of microscopic factors, such as fluid–fluid interaction, fluid–solid interaction, molecular

organization of the fluids, and molecular structure of the wall. Hence  $M$  and  $\Gamma$  may be physically related in any given system.

## 6. Concluding remarks

There are inadequacies in our present formulation. First, the free energy used to delineate the two fluids is a minimal model. It neglects, for example, the density variation that can be fairly significant in the interfacial region. Second, our model in its present form is only applicable to simple liquids. Complex fluids would require non-trivial extensions. Third, we have neglected the long-range van der Waals interaction between the wall and fluid which is very important in governing the wetting dynamics when the liquid film is on the sub-micrometer scale and subject to a gradient of disjoining pressure (de Gennes 1985). These and other inadequacies represent tasks still to be done. The main purpose of this paper is to outline the framework of a general theory which can resolve the MCL problem in its simplest form.

It is important to emphasize that while the partial slip in single-phase flows is generally small and quantitatively indistinguishable from no slip, its significance is qualitatively much greater. First, it is clear from the above that the GNBC goes hand-in-hand with the NBC in single-phase flows, and that it is incompatible with the no-slip boundary condition even in single-phase flows. The latter is clear from the power-law partial slip which extends mesoscopic distances into single-phase flow regimes. Second, even if the slip is small, the fact that slip exists means that its magnitude may be manipulated, i.e. it can be made larger or smaller. In particular, since the slip coefficient is a thermodynamic quantity, just like the viscosity, its magnitude should depend on molecular interactions and interface geometries (as well as the state variables such as temperature) (Barrat & Bocquet 1999*b*; Leger 2003; Granick, Zhu & Lee 2003; Zhu & Granick 2004; Neto *et al.* 2005), a fact which can be used to advantage experimentally through nanoscale manipulations and environmental controls. For example, effective slip at nano-patterned surfaces has already been studied (Philip 1972; Lauga & Stone 2003; Cottin-Bizonne *et al.* 2003, 2004; Priezjev, Darhuber & Troian 2005; Qian, Wang & Sheng 2005). In contrast, a no-slip boundary condition is a clear-cut statement, with no room for adjustment or for physics considerations, only carrying with it the burden of proof. Here the broad applicability of the no-slip boundary condition cannot be considered as proof against slipping, as a very small amount of partial slip would clearly lead to similar results. It is therefore rather obvious that whereas slip/partial slip can be derived from general principles and demonstrated through MD simulations, no slip has yet to be proved with similar generalities. In fact, at present slip is already a subject with a fairly extensive literature. We refer to the reviews by Granick *et al.* (2003) and by Neto *et al.* (2005) for a more complete list of references.

In closing, we note that in the case of the MCL, complete slip occurs in the linear regime (i.e.  $\beta$  is a constant). This is in contrast to the view that complete slip can only occur when ‘interface fracturing’ occurs, i.e. in the fully nonlinear regime (sometimes also denoted as ‘super slipping’ or ‘threshold slipping’), corresponding to a stress-dependent  $\beta$  (see e.g. Thompson & Troian 1997). It is likely that a statistical mechanical study of the slip coefficient can indeed produce a threshold behaviour, e.g.  $\beta$  approaches zero as the shear stress exceeds a certain threshold. However, in the linear regime this is not the case. The difference between the MCL complete slip and the interface fracturing is that in the case of MCL, the (complete) slip velocity can be very small and the relevant shear stress can be low. The fact that complete slip can

occur in this case is due to the localized nature of the uncompensated Young stress, in addition to the tangential viscous stress. Thus we can have complete slip in both the linear and nonlinear regimes, with different underlying physics.

The authors acknowledge helpful discussions with Weinan E, Chun Liu and Weiqing Ren. We would like to thank Stephen H. Davis and Masao Doi for their constructive comments. This work was partially supported by the Hong Kong RGC CERG No. 604803, the RGC central allocation grant CA05/06.SC01, and the Croucher Foundation Grant Z0138.

## Appendix A. The principle of minimum energy dissipation

To outline the principle of minimum energy dissipation (Onsager 1931*a, b*), consider a system described by one single variable  $\alpha$ , governed by the overdamped Langevin equation

$$\gamma \dot{\alpha} = -\frac{\partial F(\alpha)}{\partial \alpha} + \zeta(t), \quad (\text{A } 1)$$

where  $\gamma$  is the frictional coefficient,  $\dot{\alpha}$  is the rate of change of  $\alpha$ ,  $F(\alpha)$  is the free energy, and  $\zeta(t)$  is a white noise satisfying  $\langle \zeta(t)\zeta(t') \rangle = 2\gamma k_B T \delta(t-t')$ , with  $k_B$  denoting the Boltzmann constant and  $T$  the temperature. The probability of finding the system in the state described by  $\alpha$  is a function of time, denoted by  $P(\alpha, t)$  and governed by the Fokker–Planck equation

$$\frac{\partial P}{\partial t} = D \left[ \frac{\partial^2 P}{\partial \alpha^2} + \frac{1}{k_B T} \frac{\partial}{\partial \alpha} \left( \frac{\partial F}{\partial \alpha} P \right) \right], \quad (\text{A } 2)$$

where  $D$  is the diffusion coefficient satisfying the Einstein relation  $\gamma D = k_B T$ . It is clear that the Boltzmann distribution  $P_{eq}(\alpha) \propto \exp[-F(\alpha)/k_B T]$  is a stationary solution of the Fokker–Planck equation (A 2). It can be shown (Langer 1968) that the transition probability from  $\alpha$  at  $t$  to  $\alpha'$  at  $t + \Delta t$ , i.e.  $P_2(\alpha', t + \Delta t; \alpha, t)$ , is given by

$$P_2(\alpha', t + \Delta t; \alpha, t) = \frac{1}{\sqrt{4\pi D \Delta t}} \exp \left[ -\frac{(\alpha' - \alpha)^2}{4D \Delta t} \right] \exp \left[ -\frac{F(\alpha') - F(\alpha)}{2k_B T} \right], \quad (\text{A } 3)$$

for  $\alpha'$  in the vicinity of  $\alpha$  and short time interval  $\Delta t$ . The most probable transition occurring between  $\alpha$  and  $\alpha'$  is the one which minimizes

$$A = \frac{\gamma(\alpha' - \alpha)^2}{2\Delta t} + [F(\alpha') - F(\alpha)] \approx \left[ \frac{\gamma}{2} \dot{\alpha}^2 + \frac{\partial F(\alpha)}{\partial \alpha} \dot{\alpha} \right] \Delta t, \quad (\text{A } 4)$$

Here  $\dot{\alpha} = (\alpha' - \alpha)/\Delta t$  and the minimum of  $A$  is taken with respect to  $\alpha'$ , or equivalently  $\dot{\alpha}$ , for prescribed  $\alpha$ . The Euler–Lagrange equation for minimizing  $A$  is thus

$$\gamma \dot{\alpha} = \frac{\gamma(\alpha' - \alpha)}{\Delta t} = -\frac{\partial F(\alpha)}{\partial \alpha}, \quad (\text{A } 5)$$

as expected from the Langevin equation (A 1). Equation (A 5) is the simplest, one-variable version of the linear relation (3.1) for rates and forces, and the function  $\gamma \dot{\alpha}^2/2 + (\partial F/\partial \alpha) \dot{\alpha}$  in  $A$  is the corresponding one-variable version of the function  $\Phi(\dot{\alpha}, \alpha) + \dot{F}(\alpha, \dot{\alpha})$  in Onsager's variational principle, stated by equations (3.2), (3.3), and (3.4). From the above discussion, it is clear that (i) the variation of  $A$  should be taken with respect to the rate  $\dot{\alpha}$ , for prescribed state variable  $\alpha$ , (ii) the minimum dissipation principle implies the balance of dissipative force and the force derived from the free energy (see equation (A 5)), and (iii) the minimum dissipation principle



yields the most probable course of a dissipative process, provided the displacement from equilibrium is small (Onsager 1931*b*; Onsager & Machlup 1953).

**Appendix B. The dissipation function in steady state**

In a steady state with  $\partial\phi/\partial t = 0$ , the total rate of energy dissipation  $R_2 = 2\Phi$  in equations (3.21) and (3.22) can be expressed as a functional of  $\mathbf{v}(\mathbf{r})$  only. This is because for prescribed phase field  $\phi$ , the rate  $\mathbf{J}$  in the bulk can be determined from  $\dot{\phi} = \mathbf{v} \cdot \nabla\phi$  through equations (3.28) and (3.29) while the rate  $\dot{\phi} = v_\tau \partial_\tau \phi$  at the solid surface is already given in terms of  $\mathbf{v}$ . By expressing  $R_2$  as a functional of  $\mathbf{v}(\mathbf{r})$ , we can obtain a form of the dissipation function that is useful to the variational analysis of the sharp interface limit and the diffuse interface limit.

From equation (3.29) with  $\partial\phi/\partial t = 0$ , we can formally express  $\mu(\mathbf{r})$  by

$$\mu(\mathbf{r}) = \frac{1}{M} \int d\mathbf{r}' G(\mathbf{r}, \mathbf{r}') (\mathbf{v} \cdot \nabla\phi)_{\mathbf{r}'}, \tag{B1}$$

where  $G(\mathbf{r}, \mathbf{r}')$  is the Green function for the Laplacian operator satisfying the boundary condition  $\partial_n \mu = 0$  at the solid surface. From  $R_d$  in equation (3.19) and  $\mathbf{J} = -M\nabla\mu$  (equation (3.28)), we have

$$R_d = \int d\mathbf{r} [M(\nabla\mu)^2] = \int d\mathbf{r} [-M\mu\nabla^2\mu], \tag{B2}$$

where the integration by parts has been used with  $\partial_n \mu = 0$  at the solid surface. Substituting equation (B1) into (B2) yields

$$R_d[\mathbf{v}] = \int d\mathbf{r}_1 d\mathbf{r}_2 \left[ -\frac{1}{M} G(\mathbf{r}_1, \mathbf{r}_2) (\mathbf{v} \cdot \nabla\phi)_{\mathbf{r}_1} (\mathbf{v} \cdot \nabla\phi)_{\mathbf{r}_2} \right]. \tag{B3}$$

Substituting  $\dot{\phi} = v_\tau \partial_\tau \phi$  into  $R_r$  in equation (3.20), we obtain

$$R_r[\mathbf{v}] = \int dS \left[ \frac{1}{F} (v_\tau \partial_\tau \phi)^2 \right]. \tag{B4}$$

Combining equations (3.9), (3.10), (B3), and (B4) with  $R_2$  defined in equations (3.21) and (3.22), we obtain

$$\begin{aligned} R_2[\mathbf{v}] &= R_v[\mathbf{v}] + R_s[\mathbf{v}] + R_d[\mathbf{v}] + R_r[\mathbf{v}] \\ &= \int d\mathbf{r} \left[ \frac{\eta}{2} (\partial_i v_j + \partial_j v_i)^2 \right] + \int dS [\beta (v_\tau^{slip})^2] \\ &\quad + \int d\mathbf{r}_1 d\mathbf{r}_2 \left[ -\frac{1}{M} G(\mathbf{r}_1, \mathbf{r}_2) (\mathbf{v} \cdot \nabla\phi)_{\mathbf{r}_1} (\mathbf{v} \cdot \nabla\phi)_{\mathbf{r}_2} \right] + \int dS \left[ \frac{1}{F} (v_\tau \partial_\tau \phi)^2 \right], \end{aligned} \tag{B5}$$

for the total rate of dissipation in two-phase flows, which is the sum of  $R_v$  due to viscosity,  $R_s$  due to slipping,  $R_d$  due to diffusion in the bulk, and  $R_r$  due to relaxation at the surface.

In accordance with the principle of minimum energy dissipation (Onsager 1931*a, b*), for steady state the equation(s) of motion can be derived from minimizing the dissipation function  $\Phi$  ( $= \frac{1}{2}R_2$  here) with respect to the rates. Note that in steady state, the rates  $\mathbf{J}$  in the bulk and  $\dot{\phi}$  at the solid surface are already determined by  $\mathbf{v}$  for prescribed  $\phi$ , and  $R_2[\mathbf{v}]$  in equation (B5) displays a symmetric, quadratic form as a function(al) of  $\mathbf{v}$ , in accordance with the reciprocal relations, i.e.  $\rho_{ij} = \rho_{ji}$  for the

coefficients  $\rho_{ij}$  in  $\Phi(\dot{\alpha}, \dot{\alpha}) = \frac{1}{2} \sum_{i,j} \rho_{ij} \dot{\alpha}_i \dot{\alpha}_j$  (equation (3.3)). The variation of the total dissipation  $R_2[\mathbf{v}]$  should be taken with respect to  $\mathbf{v}$  only, as it is the only rate.

Based on equation (B 5), minimizing  $R_2$  with respect to  $\mathbf{v}$  in the bulk yields the Stokes equation (3.31), while minimizing  $R_2$  with respect to tangential fluid velocity  $v_\tau$  at the solid surface yields the GNBC (3.32). That is, consider a variation of the velocity field  $\mathbf{v}(\mathbf{r}) \rightarrow \mathbf{v}(\mathbf{r}) + \delta\mathbf{v}(\mathbf{r})$ . The associated changes in  $R_v$  and  $R_s$  are already given by equations (3.11) and (3.12), and those in  $R_d[\mathbf{v}]$  and  $R_r[\mathbf{v}]$  are given by

$$\delta R_d = \int d\mathbf{r} d\mathbf{r}' \left[ -\frac{2G(\mathbf{r}, \mathbf{r}') (\mathbf{v} \cdot \nabla \phi)_{\mathbf{r}'} (\delta\mathbf{v} \cdot \nabla \phi)_{\mathbf{r}}}{M} \right] = -2 \int d\mathbf{r} [\mu \partial_i \phi \delta v_i], \quad (\text{B } 6)$$

and

$$\delta R_r = \int dS \left[ \frac{2}{\Gamma} (v_\tau \partial_\tau \phi) \delta v_\tau \partial_\tau \phi \right] = -2 \int dS [L(\phi) \partial_\tau \phi \delta v_\tau], \quad (\text{B } 7)$$

where equations (3.29) and (3.30) have been used. Combining equations (3.11), (3.12), (3.13), (B 6), and (B 7), we obtain the Euler–Lagrange equations

$$-2\eta \partial_j (\partial_j v_i + \partial_i v_j) - \partial_i \alpha - 2\mu \partial_i \phi = 0 \quad (\text{B } 8)$$

in the bulk and

$$2\eta (\partial_n v_\tau + \partial_\tau v_n) + 2\beta v_\tau^{\text{slip}} - 2L(\phi) \partial_\tau \phi = 0 \quad (\text{B } 9)$$

at the surface. Note that equation (B 8) is identical to the Stokes equation (3.31) with  $\partial_j v_j = 0$  and  $\alpha = -2p$ , and equation (B 9) reduces to the GNBC (3.32).

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